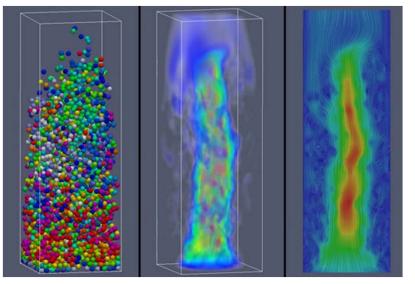
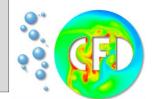


Finite Element-Fictitious Boundary Methods for the Numerical Simulation of Complex Particulate Flows



Stefan Turek, Raphael Münster, Otto Mierka Institut für Angewandte Mathematik, TU Dortmund <u>http://www.mathematik.tu-dortmund.de/LS3</u> <u>http://www.featflow.de</u>



Basic Flow Solver

Basic CFD tool – FEATELOW (robust, parallel, efficient)

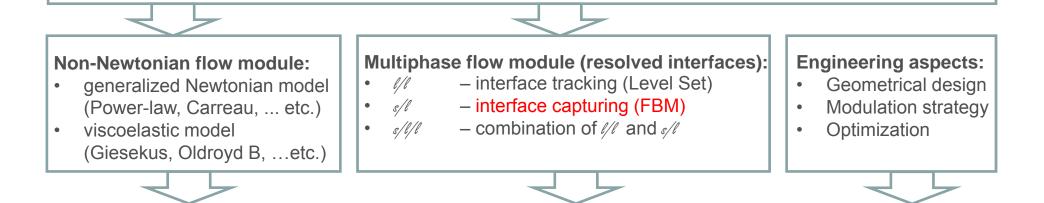
HPC features:

- Massively parallel
- GPU computing
- Open source

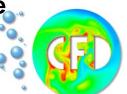


Numerical features:

- Higher order Q2P1 FEM schemes
- FCT & EO FEM stabilization techniques
- Use of unstructured meshes
- Fictitious Boundary (FBM) methods
- Dynamic adaptive grid deformation
- Newton-Multigrid solvers



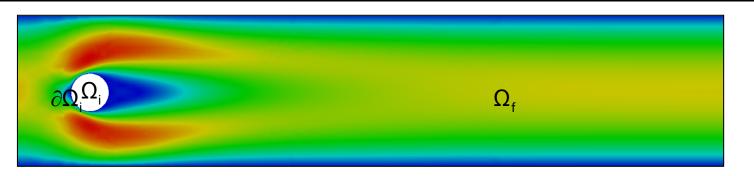
FEM-based simulation tools for the accurate prediction of multiphase. flow problems, particularly with liquid-solid interfaces



Liquid– (Rigid) Solid Interfaces



Consider the flow of N solid particles in a fluid with density ρ and viscosity μ . Denote by $\Omega_f(t)$ the domain occupied by the fluid at time t, by $\Omega_i(t)$ the domain occupied by the ith-particle at time t and let $\overline{\Omega} = \overline{\Omega}_f \cup \overline{\Omega}_i$.



The fluid flow is modelled by the Navier-Stokes equations:

$$\rho \left(\frac{\partial u}{\partial t} + u \cdot \nabla u \right) - \nabla \cdot \sigma = f, \quad \nabla \cdot u = 0$$

where σ is the total stress tensor of the fluid phase:

$$\sigma(\mathbf{X},t) = -p\mathbf{I} + \mu[\nabla u + (\nabla u)^{\mathsf{T}}]$$





The motion of particles can be described by the **Newton-Euler equations**. A particle moves with **a translational velocity** U_i and **angular velocity** ω_i which satisfiy:

$$M_{i}\frac{dU_{i}}{dt} = F_{i} + F_{i}' + (\Delta M_{i})g, \qquad I_{i}\frac{d\omega_{i}}{dt} + \omega_{i} \times (I_{i}\omega_{i}) = T_{i,}$$

- M_i : mass of the i-th particle (i=1,...,N)
- I_i : moment of inertia tensor of the i-th particle
- ΔM_i : mass difference between M_i and the mass of the fluid
- F_i : hydrodynamic force acting on the i-th particle
- T_i : hydrodynamic torque acting on the i-th particle





The position and orientation of the i-th particle are obtained by integrating the **kinematic equations**:

$$\frac{dX_{i}}{dt} = U_{i}, \ \frac{d\theta_{i}}{dt} = \omega_{i}, \ \frac{d\omega_{i}}{dt} = I_{i}^{-1}T_{i}$$

which can be done numerically by an explicit Euler scheme:

$$X_{i}^{n+1} = X_{i}^{n} + \Delta t U_{i}^{n} \quad \omega_{I}^{n+1} = \omega_{I}^{n} + \Delta t \left(I_{i}^{-1}T_{i}^{n}\right) \quad \theta_{I}^{n+1} = \theta_{I}^{n} + \Delta t \omega_{I}^{n}$$

Boundary Conditions

We apply the velocity u(X) as no-slip boundary condition at the interface $\partial \Omega_i$ between the i-th particle and the fluid, which for $X \in \Omega_i$ is defined by:

$$u(X) = U_i + \omega_i \times (X - X_i)$$

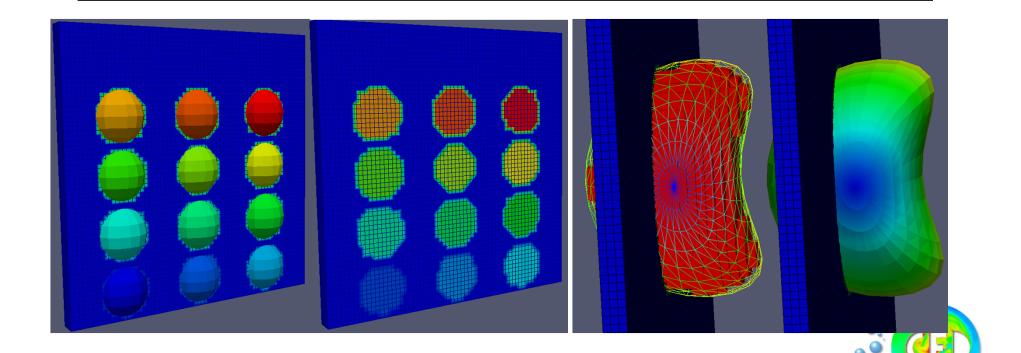


Fictitious Boundary Method



Eulerian Approach:

- Internal objects are represented as a boolean (in/out) function on the mesh
- Use of a fixed mesh possible
- Complex shapes are possible (surface triangulation, implicit functions)
- Higher accuracy possible by using mesh adaptation techniques



Hydrodynamic Forces



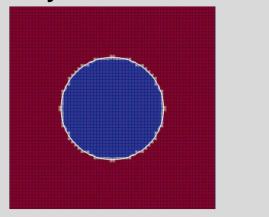
Hydrodynamic force and torque acting on the i-th particle

$$F_{i} = -\int_{\partial\Omega_{i}} \sigma \cdot n_{i} d\Gamma_{i}, \quad T_{i} = -\int_{\partial\Omega_{i}} (X - X_{i}) \times (\sigma \cdot n_{i}) d\Gamma$$

Force Calculation with Fictitious Boundary Method

The FBM can only decide:

- `INSIDE`(1) and `OUTSIDE`(0)
- Only first order accuracy



Alternative: Replace the surface integral by a volume integral



Define an *indicator function* α_i :

$$\alpha_{i}(X) = \begin{cases} 1 & \text{for } X \in \Omega_{i} \\ 0 & \text{for } X \in \Omega_{f} \end{cases}$$

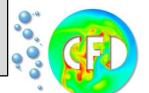
Remark: The gradient of α_i is zero everywhere except at the surface of the i-th Particle and approximates the normal vector (in a weak sense), allowing us to write:

$$\mathsf{F}_{i} = -\int_{\Omega T} \sigma \, \cdot \nabla \, \alpha_{\,i} \mathrm{d} \, \Omega \ , \quad \mathsf{T}_{i} = -\int_{\Omega T} (\mathsf{X} - \mathsf{X}_{\,i}) \times \big(\sigma \cdot \nabla \, \alpha_{\,i} \big) \mathrm{d} \, \Omega$$

On the finite element level we can compute this by:

$$\begin{aligned} \mathsf{F}_{i} &= -\sum_{\mathsf{T} \in \mathsf{T}_{h,i}} \int_{\Omega_{\mathsf{T}}} \sigma_{\mathsf{h}} \cdot \nabla \alpha_{\mathsf{h},i} d\Omega , \\ \mathsf{T}_{i} &= -\sum_{\mathsf{T} \in \mathsf{T}_{h,i}} \int_{\Omega_{\mathsf{T}}} \left(\mathsf{X} - \mathsf{X}_{\mathsf{i}} \right) \times \left(\sigma_{\mathsf{h}} \cdot \nabla \alpha_{\mathsf{h},i} \right) d\Omega \end{aligned}$$

 $\alpha_{h,i}(x)$: finite element interpolant of $\alpha(x)$ T_{h,i} : elements intersected by i-th particle



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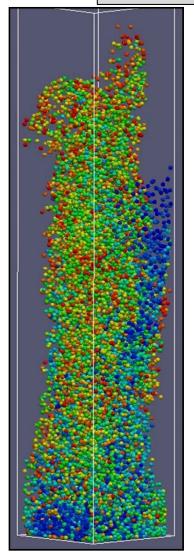
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Numerical Force Evaluation (II)



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Large-scale FBM-Simulations



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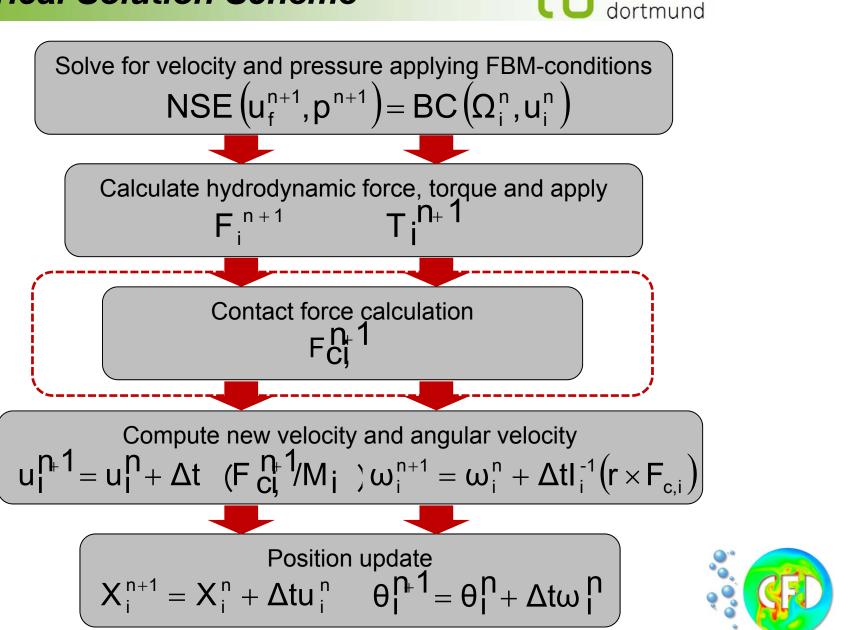
Integration over Ω_T too expensive:

- Gradient is non-zero on $\partial \Omega_i$
- Information available from FBM
- Evaluate boundary cells only
- Visit each cell only once

$$- \int_{T_{i}} T_{i} = -\sum_{T \in T_{h,i}} \int_{\Omega_{T}} (X - X_{i}) \times (\sigma_{h} \cdot \nabla \alpha_{h,i}) d\Omega$$



Numerical Solution Scheme

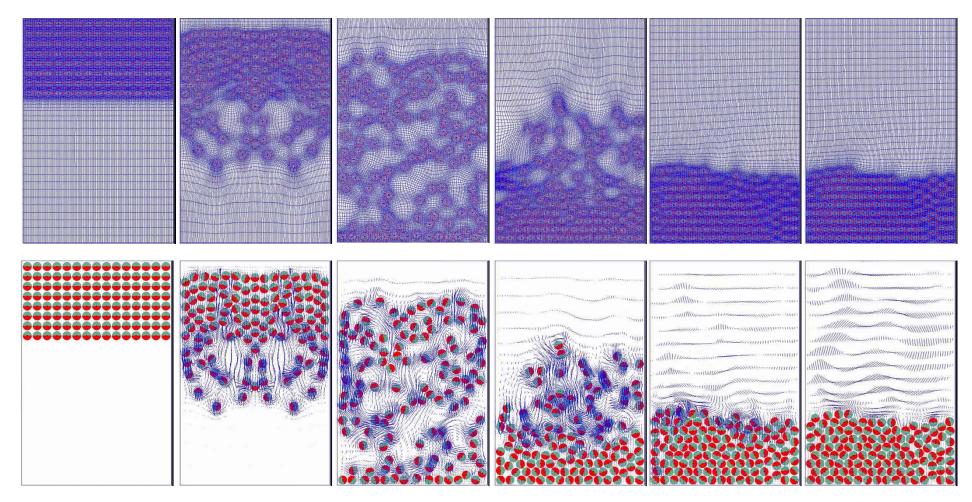


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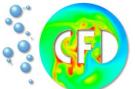
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Grid Deformation Method



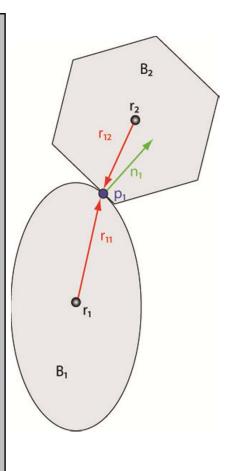


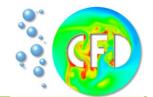
Further improvement via adaptive Grid Deformation which preserves the (local) logical structure (\rightarrow GPU)



Contact Force Calculation

- Contact force calculation realized as a three step process
 - \rightarrow Broadphase
 - → Narrowphase
 - → Contact/Collision force calculation
- Worst case complexity for collision detection is $O(n^2)$
 - → Computing contact information is expensive
 - → Reduce number of expensive tests → Broad Phase
- Broad phase
 - \rightarrow Simple rejection tests exclude pairs that cannot intersect
 - → Use hierarchical spatial partitioning
- Narrow phase
 - → Uses Broadphase output
 - → Calculates data neccessary for collision force calculation





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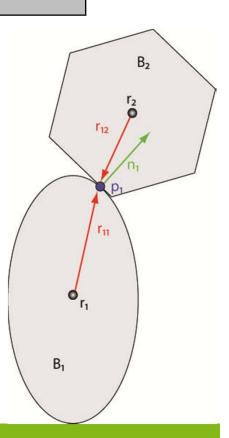
Single Body Collision Model

For a single pair of colliding bodies we compute the impulse f that causes the velocities of the bodies to change:

$$f = \frac{-(1 + \epsilon)(n_1(v_1 - v_2) + \omega_1(r_{11} \times n_1) - \omega_2(r_{12} \times n_1))}{m_1^{-1} + m_2^{-1} + (r_{11} \times n_1)^T I_1^{-1}(r_{11} \times n_1) + (r_{12} \times n_1)^T I_2^{-1}(r_{12} \times n_1)}$$

Using the impulse f, the change in linear and angular velocity can be calculated:

$$v_{1}(t + \Delta t) = v_{1}(t) + \frac{fn_{1}}{m_{1}}, \omega_{1}(t + \Delta t) = \omega_{1}(t) + I_{1}^{-1}(r_{11} \times fn_{1})$$
$$v_{2}(t + \Delta t) = v_{2}(t) - \frac{fn_{1}}{m_{2}}, \omega_{2}(t + \Delta t) = \omega_{2}(t) - I_{2}^{-1}(r_{12} \times fn_{1})$$



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Multi-Body Collision Model



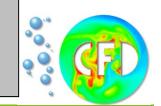
In the case of **multiple colliding bodies** with *K* contact points the impulses influence each other. Hence, they are combined into a system of equations that involves the following matrices and vectors:

- N: matrix of contact normals
- C: matrix of contact conditions
- M: rigid body mass matrix
- f: vector of contact forces (f_i≥0)
- f^{ext}: vector of external forces(gravity, etc.)

$$\frac{N^{\mathsf{T}}C^{\mathsf{T}}M^{-1}CN}{A} \cdot \frac{\Delta tf^{t+\Delta t}}{x} + \frac{N^{\mathsf{T}}C^{\mathsf{T}}\left(u^{t} + \Delta tM^{-1} + f^{ext}\right)}{b} \ge 0, f \ge 0$$

A problem of this form is called a **linear complementarity problem** (LCP) which can be solved with efficient iterative methods like the **Projected Gauss-Seidel solver (PGS)**.

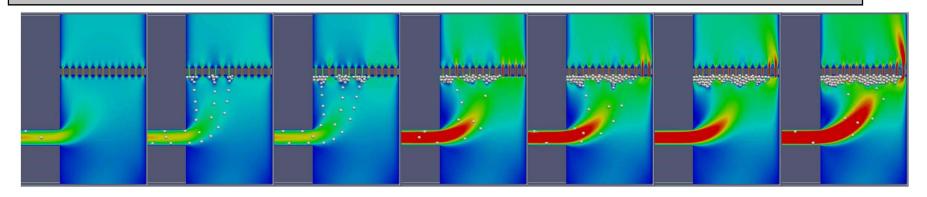
Kenny Erleben, Stable, Robust, and Versatile Multibody Dynamics Animation

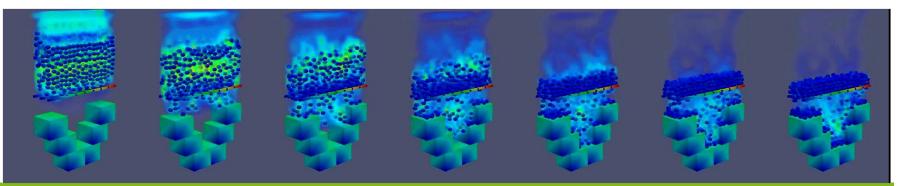


Examples



- Fluidized bed
- Particulate flow demonstrating incompressibility
- GPU sedimentation example
- Numerical results and benchmark test cases
- Comparison of results with other groups





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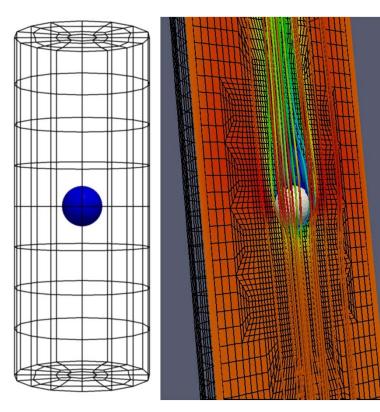
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Benchmarking and Validation (I)

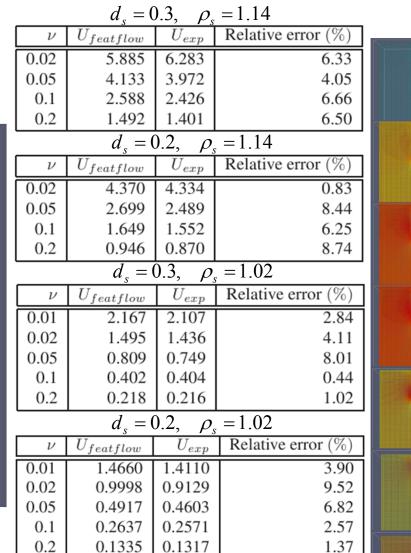


Free fall of particles:

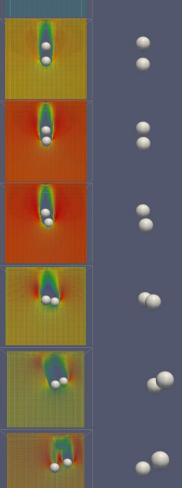
- Terminal velocity
- Different physical parameters
- Different geometrical parameters



Münster, R.; Mierka, O.; Turek, S.: Finite Element fictitious boundary methods (FEM-FBM) for 3D particulate flow, IJNMF, 2011



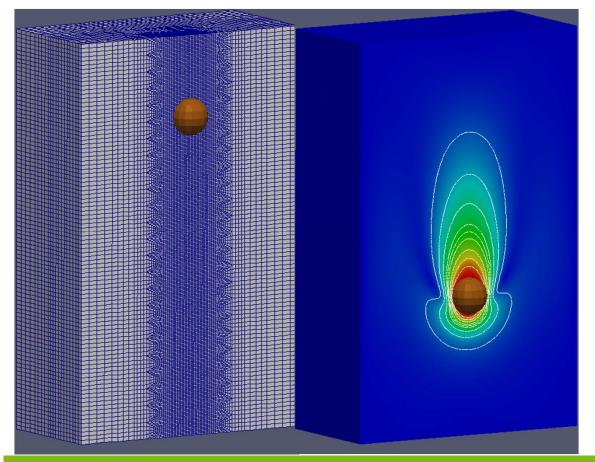
Source: Glowinski et al. 2001



Benchmarking and Validation (II)

Settling of a sphere towards a plane wall:

- Sedimentation Velocity
- Particle trajectory
- Kinetic Energy
- Different Reynolds numbers



Setup

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Computational mesh:

- 1.075.200 vertices
- 622.592 hexahedral cells
 O2/P1
 - Q2/P1:

ΠJ

→ 50.429.952 DoFs

Hardware Resources:

32 Processors

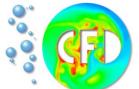


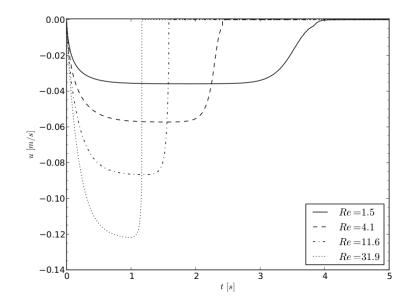
Observations

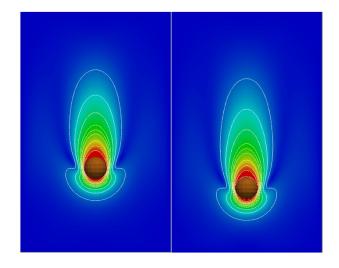
- Velocity profiles compare well to ten Cate's data
- Maximum velocity close to experiment •
- Flow features are accurately resolved

Re	u_{max}/u_{∞}	u_{max}/u_{∞}	u_{max}/u_{∞}
		ten Cate	exp
1.5	0.945	0.894	0.947
4.1	0.955	0.950	0.953
11.6	0.953	0.955	0.959
31.9	0.951	0.947	0.955

Tab. 1 Comparison of the u_{max}/u_{∞} ratios between the FEM-FBM, ten Cate's simulation and ten Cate's experiment



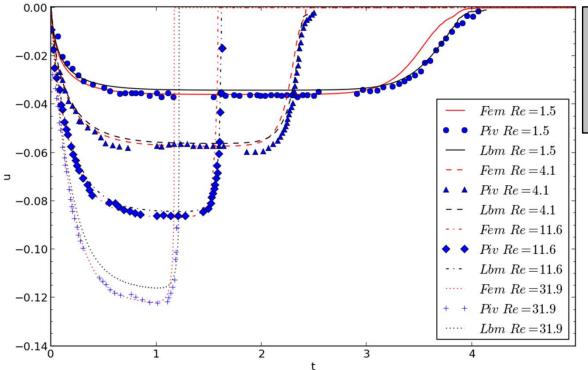




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Comparison





Comparison of FEM-FBM and the experimental values and the LBM results of the group of Sommerfeld

Source: 13th Workshop on Two-Phase Flow Predictions 2012 Acknowledgements: Ernst, M., Dietzel, M., Sommerfeld, M.

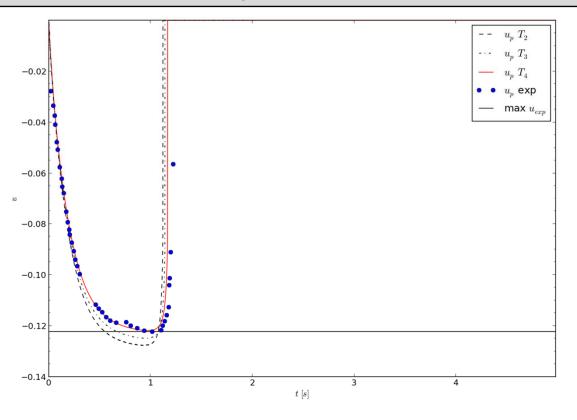


Multi-level Analysis



FEM-Multigrid Framework

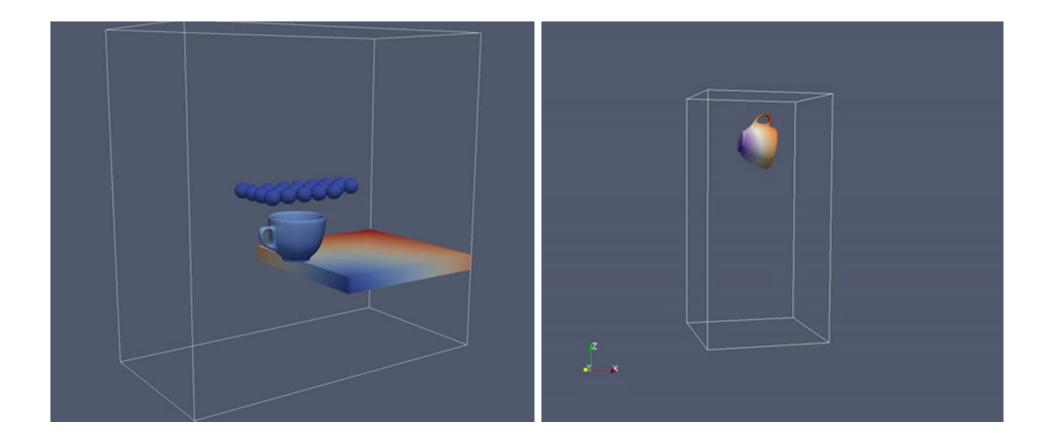
- Increasing the mesh resolution produces more accurate results Test performed at different mesh levels
 - Maximum velocity is approximated better
 - Shape of the velocity curve matches better





Complex Geometry Examples

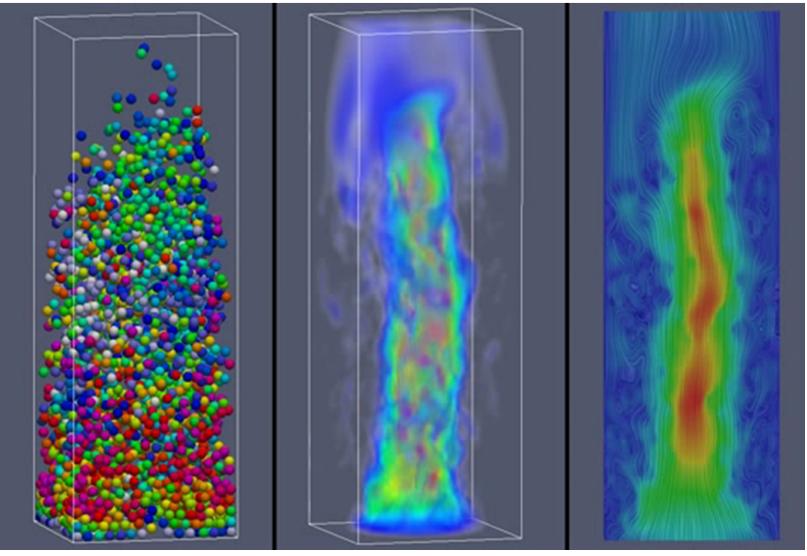






Fluidized Bed Example

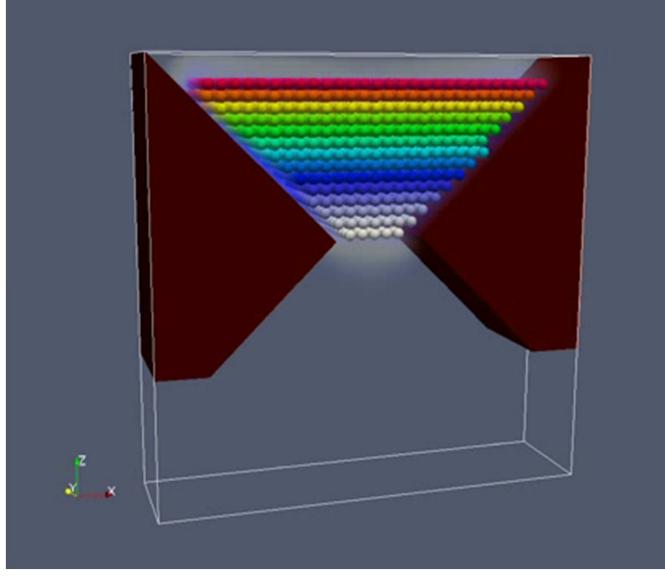






DGS Configuration





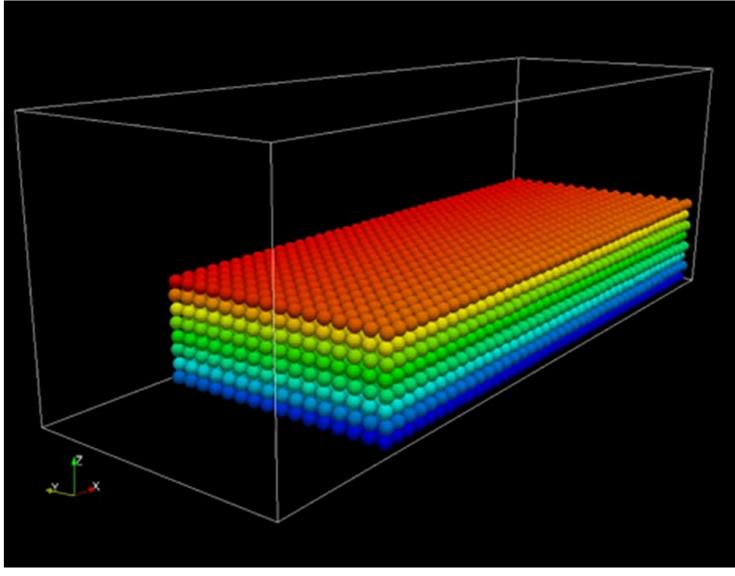


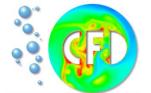
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Large-Scale Examples

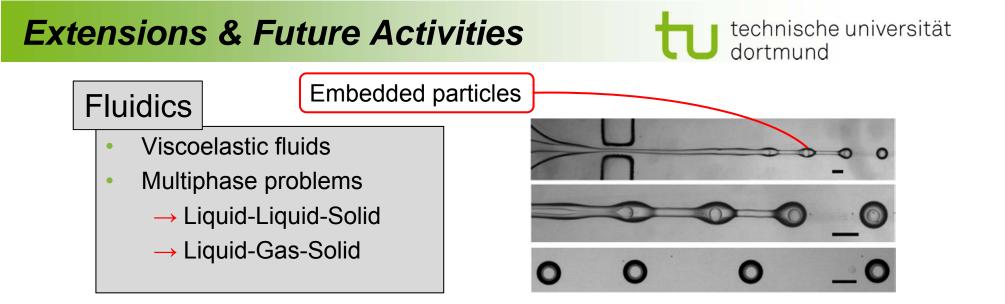






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Hardware-Oriented Numerics

- Improve parallel efficiency of collision force computation
- Further develop collision detection and collision force computation on GPUs

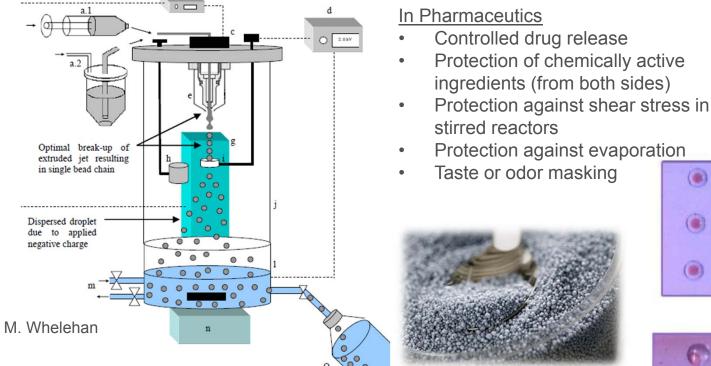






Fluid Prilling and Encapsulation (I)

- Numerical simulation of micro-fluidic drug encapsulation ("monodisperse compound droplets")
- Polymeric "bio-degradable" outer fluid with generalized Newtonian behaviour
- Optimization w.r.t. boundary conditions, flow rates, droplet size, geometry



Jet Configuration

- Core material is defined as the specific material that requires to be coated (liquid, emulsion, colloid or solid)
- Shell material is present to protect and stabilize the core (Alginate, Chitosan, Gelatin, Pectin, Waxes, Starch)

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